(54) HEAT EXCHANGER

(11) 3-286996 (A)

(43) 17.12.1991 (19) JP

(21) Appl. No. 2-86595 (22) 30.3.1990

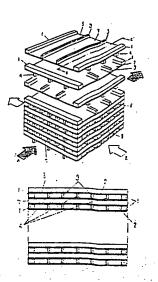
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(51) Int. Cl<sup>5</sup>. F28F3/04,F24F7/08

PURPOSE: To simplify a manufacturing process and to reduce manufacturing cost by integrally molding a plurality of the ribs provided on the surface of a heat transfer plate and a plurality of the ribs provided on the rear thereof so as to cross said ribs at a right angle from a resin so as to hold the heat transfer plate therebetween to form a heat exchange plate and laminating a plurality of the heat exchange plates

so as to mutually shift them by 90°.

CONSTITUTION: A heat exchange plate 5 is formed by integrally molding two shielding ribs 1 and two spacer ribs 3 provided on the surface of a heat transfer plate 2 and the holding ribs 4 provided on the rear thereof from a resin so as to hold the heat transfer plate 2 therebetween. A plurality of the heat exchange plates 5 are laminated while alternately shifted by 90° so that the holding ribs 4 provided on the rears of the adjacent heat transfer plates 2 are positioned between the spacer ribs 3 provided on the surfaces of the heat transfer plates 2 to form a heat exchanger 6 having ventilation paths 7 allowing a primary air stream to flow and ventilation paths 8 allowing a secondary air stream to flow provided every one layer. Since the manufacturing process of the heat exchanger 6 consists of two processes, that is, a process for the integral molding of the heat exchange plate 5 by a molding machine and a process for laminating the heat transfer plates while mutually shifting them by 90°, manufacturing cost can be reduced.



19 日本国特許庁(JP)

⑩ 特許出願公開

平3-286996

四条合为中籍新陈玉は藤瀬建筑治別区

平成3年(1991)12月17日

F 28 F F 24 F 7/08

1 0 1 A

6925-3L

請求項の数 1 (全4頁)

三 ※ 5 ※ 20特 高願 = 平2-86595

医多类性 多路線

4 多為第 @出 3 顧 平 2 (1990) 3 月30日

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9代

1、発明の名称

熱交換器

2、特許請求の範囲

正方形の伝熱板表面の向かい合う端部に設け た、伝熱板の一辺と同寸法の2本の遠へいりで と、上記述へいりブの間に所定間隔で複数本設け た間隔リブと、上記伝熱板裏面に上記間隔リブと 直交し、かつ所定間隔で複数本数けた保持リプと を、上記伝熱板を間にはさんで樹脂にて一体に成 型して熱交換板とし、この熱交換板を交互に90 度すらしながら、かつ熱交換板の伝熱板表面に設 けた間隔リブと間隔リブの間に、繰り合う熱交換 板の伝熱板裏面に設けた保持リプを位置させて複 数枚費用心大熱交換器生产之一可以多位58時度 3.5. 英明の詳細な説明など、パズムショウならころす 産業上の利用分野をいるがら別はなっ ダース 明は熱交換形換気扇等に使用す に関するものであることのははいません

従来、この種の熱交換器は第3図および第4図 に示すように、紙、あるいはプラスチックの薄板 状の伝熱板100と、波形の間隔板101とを貼 り合わせて熱交換板102とし、この熱交換板102 を交互に90度すらしなか ら複数枚積層して熱交 換器103を形成し、1次気流Xと2次気流Yの 間で熱交換をおこなう構成となっていた。

発明が解決しようとする課題

このような従来の構成では、熱交換器103の 製造工程が、間隔板101の段折り→伝熱板100 と間隔板102の貼合による熱交換板102の作 成→熱交換器板102の切断→發層→熱交換器103 の所定寸法となるよう仕上げ切断、となってお り、製造コストが高くなっていた。また、積層し た後での任主げ切断では間隔板101の目がつぶ れやすぐい切断作業が困難であった。また、 然交換器103を熱交換形換 気扇等に使用した場 今、間隔板101の板厚により、伝熱板100に て形成される通風路の有効 面積 が小さくなるた め、抵抗損失が大きく、高静圧形の送風機が必要

となっていた。また、一般的に全熱交換用として 伝熱板100と間隔板101を紙にて製造するが この場合、熱交換器103は非常に壊れやすく、 清掃時に間隔板102の目をつぶしたり、落下時 に破損したりする恐れがあり、また長期間の足形 により、伝熱板100や間隔板101が吸湿や乾 爆を繰り返して収縮し、1次気流とて2次気流が 提合しやすくなるなど耐久性に問題があった。

 とするものである。

課題を解決するための手段

### 作用

この構成により、伝熱板表面に設けた間隔リブと関り合う伝熱板裏面に設けた保持リブとが互いの伝熱板を保持し合うことになり、1次気流と2次気流の流れる通風路が一層おきに安定して形成されることとなる。

### 実 施 例

以下本発明の一実施例を第1図および第2図に もとづいて説明する。図において、1は正方形の 伝熱板2の表面の向かい合う両端部に設けた2本 の遠へいりプで、伝熱板2の一辺と同寸法となっ ている。3は伝熱板2の表面に設けた間隔リブ で、2本の速へいリプ1の間に所定間隔で複数本 設けられている。4は伝熱板2の裏面に設けた保 持リブで、間隔リブ3と直交し、かつ所定間隔で 複数本設けられている。5は熱交換板で、伝熱板 2 を間にはさみ、伝熱板 2 の表面に設けた 2 本の 遠へいリブ1と間隔リブ3と、伝熱板2の裏面に 設けた保持リブ4とを樹脂にて一体に成型したも のである。6は熱交換板5を交互に90度すらし ながら、かつ伝熱板2の表面に設けた間隔リブ3 と間隔リブ3の間に、隣り合う伝熱板2の裏面に 設けた保持リブ4を位置させて複数枚積層した熱 交換器で、1次気流を流す通風路7と2次気流を 施す通風路8とを一層おきに形成する構成となっ

上記構成において、1次気流を矢印Aのように 通風路7に流し、2次気流を矢印Bのように通風 路8に流すと、伝熱板2により1次気流と2次気 旅の熱が交換される。また、熱交換器6の製造工 程は、成型機による熱交換板5の一体成型一交互 に90度すらしなからの稜層、の2工程ですむの で製造コストが低減でき、また、熱交換板5を成 型機で成型するため寸法精度がよく、積層して熱 交換器 6 とした後で仕上げのための切断をする必 要がない。また、伝熱板2の表面に投けた間隔り プ3と間隔リプ3の間に隣り合う伝熱板2の裏面 に設けた保持リブ4を位置させて被層するため、 間隔リブ3と保持リブ4か伝熱板2を保持しあ い、1次気流を施す通風路7と2次気流を施す通 風路 8 とが安定して形成され、抵抗損失を少さく することが可能となり、したが完工業品機会からの くすることが可能となる。またお答念い見るよい 間隔リブ3、保持リブ4が側頭製のため 熱交換 器6が非常に強固なものとなり、液播時の目つぶ れかなくなり、落下等でも容易に変形することも

1.

5に本発明は、正方形の **畠部に設けた、伝熱板の** へいリアと、上記書へい 数本設けた間隔引プと、 隔りプと直交し、かつ所 持りプとを、上記伝熱板 一体に成型して熱交換板 交互に90度ずらしなが 3.板表面に設けた間隔リブ )合う熱交換板の伝熱板裏 立置させて複数枚積層した

क्रिया है। 人类为多

熱板表面に設けた間隔リブ に設けた保持リブとが互い ことになり、1次気流と2 か一層おきに安定して形成

1次気流を矢印Aのように 欠気能を矢印Bのように通風。 反2により1次気流と2次気 。また、熱交換器6の製造工。 熱交換板5の一体成型→交互 らの後層、の2工程ですむの でき、また、熱交換板5を成 寸法精度がよく、教服して熱 仕上げのための切断をする必 熱板2の表面に設けた間隔リ )間に隣り合う伝熱板2の裏面 上を位置させて被雇するため対 リズムが伝熱板2を保持し方 す通風路7と2次気施を施言機 て形成され対抵抗損失変公方系 なり、したが見る盗馬機長必さら となる。またれ無命は生産品 リブを対射服器の在的問題交換 なものとなり、意想時の長品が 下等でも容易に変形変えることも

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なく、長期使用に耐えることが可能となる。

### - 発明の効果

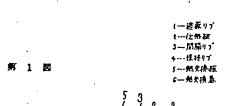
は、伝熱板表面に設けた複数のリブと、このリブ と直交して伝熱板裏面に設けた複数のリブとを、 伝熱板をはさんで樹脂にて一体に成型して熱交換 板とし、この熱交換板を交互に90度ずらしなが ら、かつ伝熱板表面のリブとリブの間に、隣り合 う伝熱板裏面のリブを位置させて複数枚積層して 熱交換器を形成することにより、製造工程を簡略 化して製造コストを低減し、樹脂製リブによって。 通風路面積を大きくして抵抗損失を小さくすると ともに熱交換器を強固なものとし、経年変化が少 なく、耐久性を向上することが可能となる等の効 果がある。

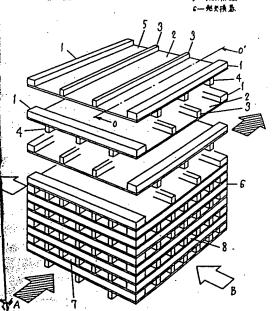
### 4、図面の簡単な説明

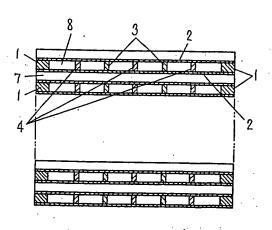
第1図は本発明の一実施例における熱交換器の 斜視図、第2図は同熱交換器の第1図の〇一〇 断面図、第3図は従来の熱交換器の組立状態を示 す料視図、第4図は同従来の熱交換器の完成品の 斜視図である。

1 …… 這へいリブ、2 … … 伝熱板、3 … … 間隔 前記実施例の説明より明らかなように本発明 リブ、4……保持リブ、5……熱交換板、6……

代理人の氏名 弁理士 栗野篤孝 ほか1名

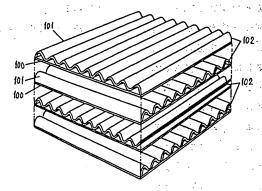


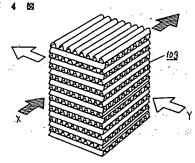




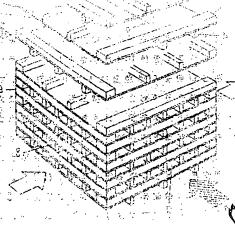
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第 3 🛭





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(2) Kokai (Japanese Unexamined Patent Publication) No. 3-286996

Title of the Invention: Heat Exchanger Publication Date: December 17, 1991

Application No. 2-86595 Filing Date: March 30, 1990

Applicant: Matsushita Precision Works, Co., Ltd.

Inventor: Shinji Ogawa

### **CLAIM**

A heat exchanger, wherein two shield ribs of the same length as one side of a square heat transmission plate are formed along the opposed end portions of the obverse surface of the heat transmission plate, wherein a plurality of spacer ribs are formed at predetermined intervals between the shield ribs, wherein a plurality of holding ribs are formed at predetermined intervals on the reverse surface of the heat transmission plate in the direction perpendicular to the spacer ribs, and wherein the shield ribs, the spacer ribs, the holding ribs and the heat transmission plate are molded into a heat exchange plate of resin, and wherein a plurality of the heat exchange plates are stacked while being displaced by 90 degrees alternately, so that the holding ribs formed on the reverse surface of the heat transmission plate of a heat exchange plate are located between the spacer ribs formed on the obverse surface of the heat transmission plate of an adjacent heat exchange plate.

3. Detailed Description of the Invention

Industrial Field of Utilization:

The present invention relates to a heat exchanger used for a ventilation fan of heat exchange type or the like.

Prior Art:

Conventionally, a heat exchanger of this type, as shown in Figs. 3 and 4, is configured of a plurality of heat exchange plates 102, each including a thin heat transmission plate 100 of paper or plastics and a corrugated spacer plate 101 attached to each other, wherein the heat exchange plates 102 are stacked while being displaced alternately by 90 degrees thereby to form a heat exchanger 103 for performing the heat exchange operation between a primary air current X and a secondary air current Y.

Problem to be Solved by the Invention:

The process of fabricating the conventional heat exchanger 103 having the configuration described above comprises the steps of corrugating the spacer plates 101, fabricating each heat exchange plate 102 by attaching the heat transmission plate 100 and the spacer plate 101 to each other, cutting the heat exchange plates 102, stacking the heat exchange plates 102, and finish cutting to complete the heat exchanger 103 of a predetermined size. This process involves a high fabrication cost. Also, the finish cutting after stacking is liable to squash the spaces of the spacer plates 101, which makes the cutting work difficult. In

the case where this heat exchanger 103 is used for a ventilation fan of heat exchange type or the like, the thickness of the spacer plates 101 reduces the effective area of the air paths formed on the heat transmission plates 100, and the resultant increased resistance loss requires a fan of a high static pressure type. Generally, the heat transmission plate 100 and the spacer plate 101 are formed of paper for total heat exchange. In this case, the heater exchanger 103 is easily broken, with the probable result that the spaces of the spacer plates 102 are squashed at the time of cleaning or ruptured by being dropped. Also, after long use, the heat transmission plates 100 and the spacer plates 101 repeatedly absorb moisture or are dried, thereby making the primary air current and the secondary air current liable to mix with each other for a reduced durability.

The present invention is intended to solve these problems of the prior art, and the object thereof is to provide a highly durable heat exchanger comprising a plurality of heat exchange plates each configured of a heat transmission plate, wherein a plurality of ribs are formed on the obverse surface of the heat transmission plate, and a plurality of ribs perpendicular to the ribs on the obverse surface of the heat transmission plate are formed on the reverse surface of the same heat transmission plate, wherein the two types of ribs with the heat transmission plate therebetween are integrally molded, using a resin material, into a heat exchange plate of resin, and wherein a plurality of the heat exchange plates are stacked while being displaced alternately by 90 degrees, so that the ribs on the reverse surface of a given heat transmission plate are located between the ribs on the obverse surface of an adjacent heat transmission plate in staggered fashion, thereby forming a heat exchanger. In this way, the fabrication process is simplified for a reduced fabrication cost, the resin ribs reduce the resistance loss by increasing the area of the air paths, and the strength of the heat exchanger is increased, while at the same time reducing variations.

Means for Solving the Problems:

In order to solve these problems, according to the invention, there is provided a heat exchanger, wherein two shield ribs of the same length as one side of a square heat transmission plate are formed along the opposed end portions of the obverse surface of the heat transmission plate, wherein a plurality of spacer ribs are formed at predetermined intervals between the shield ribs, wherein a plurality of holding ribs are formed at predetermined intervals, wherein the shield ribs, the spacer ribs, the holding ribs and the heat transmission plate are molded into a heat exchange plate of resin, and wherein a plurality of the heat exchange plates are stacked while being displaced by 90 degrees alternately, so that the holding ribs formed on the reverse surface of the heat transmission plate of a heat exchange plate are located between the spacer ribs formed on the obverse surface of the heat transmission plate of an adjacent heat exchange plate.

Operation:

As the result of this configuration, the spacer ribs arranged on the obverse surface of a heat transmission plate and the holding ribs arranged on the reverse

surface of an adjacent heat transmission plate cooperate with each other to hold the heat transmission plates. In this way, air paths for passing the primary air current and the secondary air current are formed positively and stably in alternate layers.

### Embodiments:

An embodiment of the invention will be explained below with reference to Figs. 1 and 2. In the drawings, numeral 1 designates two shield ribs of the same size as one side of a square heat transmission plate 2, which shield ribs are formed along the opposed end portions, respectively, on the obverse surface of the heat transmission plate 2. Numeral 3 designates a plurality of spacer ribs arranged on the obverse surface of the heat transmission plate 2, which spacer ribs are arranged at predetermined intervals between the two shield ribs 1. Numeral 4 designates a plurality of holding ribs arranged at predetermined intervals on the reverse surface of the heat transmission plate 2 in positions perpendicular to the spacer ribs 3. Numeral 5 designates a heat exchange plate of resin formed by integrally molding the heat transmission plate 2, the two shield ribs 1 and the spacer ribs 3 arranged on the obverse surface of the heat transmission plate 2 and the holding ribs 4 arranged on the reverse surface of the heat transmission plate 2. Numeral 6 designates a heat exchanger comprising a plurality of heat exchange plates 5 stacked by being displaced 90 degrees alternately, so that the holding ribs 4 arranged on the reverse surface of a heat transmission plate 2 are located between the spacer ribs 3 arranged on the obverse surface of an adjacent heat transmission plate 2, and air paths 7 for passing the primary air current and air paths 8 for passing the secondary air current are formed in alternate layers.

In the configuration described above, the primary air current is passed through the air paths 7 as indicated by arrow A, and the secondary air current is passed through the air paths 8 as indicated by arrow B. Then, the heat of the primary air current is exchanged with the heat of the secondary air current by the heat transmission plates 2. The heat exchanger 6 can be fabricated only through the two steps of integrally molding the heat exchange plate 5 by a molding machine, and stacking the heat exchange plates 5 while displacing them alternately by 90 degrees. Thus, the fabrication cost can be reduced. Also, since a molding machine is used to mold each heat exchange plate 5, a high dimensional accuracy can be secured. After stacking the heat exchange plates 5 into a heat exchanger 6, therefore, the process of finish cutting is not required. Further, in view of the fact that the holding ribs 4 arranged on the obverse surface of a heat transmission plate 2 are located between the spacer ribs 3 arranged on the reverse surface of an adjacent heat transmission plate 2, while stacking the heat exchange plates, the spacer ribs 3 and the holding ribs 4 cooperate with each other in holding the heat transmission plates 2. Thus, the air paths 7 for passing the primary air current and the air paths 8 for passing the secondary air current are positively and stably formed alternately. In this way, both the resistance

loss and the size of the fan can be reduced. Also, since the shield ribs 1, the spacer ribs 3 and the holding ribs 4 are formed of resin, the strength of the heat exchanger 6 is greatly increased. As a result, the spaces are prevented from being squashed at the time of cleaning, and the heat exchanger 6 is not easily deformed even if dropped, thereby lengthening the service life thereof.

Effects of the Invention:

As apparent from the foregoing description of an embodiment, according to this invention, there is provided a heat exchanger comprising a plurality of heat transmission plates, each of which includes a plurality of ribs arranged on the obverse surface thereof and a plurality of ribs arranged on the reverse surface thereof in the direction perpendicular to the ribs on the obverse surface, wherein the ribs and the heat transmission plate therebetween are integrally molded using a resin material thereby to form a heat exchange plate, wherein a plurality of the heat exchange plates are stacked while being displaced alternately by 90 degrees, so that the ribs on the reverse surface of each heat transmission plate are located between the ribs on the obverse surface of an adjacent heat transmission plate, thereby forming a heat exchanger. In this way, the fabrication process is simplified for a reduced fabrication cost, and the area of the air paths is increased by the resin ribs with a reduced resistance loss while increasing the strength of the heat exchanger. In addition, the secular variations are reduced, and the durability is improved.

### 4. Brief Description of the Drawings

Fig. 1 is a perspective view of a heat exchanger according to an embodiment of the invention, Fig. 2 a sectional view taken in line O-O' in Fig. 1, Fig. 3 a perspective view showing the manner in which the conventional heat exchanger is assembled, and Fig. 4 a perspective view of the same heat exchanger in completed form.

1...Shield rib, 2...Heat transmission plate, 3...Space rib, 4...Holding rib, 5...Heat exchange plate, 6...Heat exchanger

## Drawing

- 1...Shield rib
- 2...Heat transmission plate
  3...Space rib
  4...Holding rib
  5...Heat exchange plate
  6...Heat exchanger